

#### Winning the NIBE Project of the Year Award

**Ground Source Live 2011** 

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Groundwater Specialists Ltd www.ogi.co.uk



#### **NIBE Project Award Criteria**

- Includes NIBE Ground Source Heat Pump
- Demonstrates Innovation
- Overcoming Difficulties
- Sustainable Renewable Energy Resource
- Must provide Energy/Cost Savings.



## **The Project**

- 55 Bedroom Care Home for the Elderly
- Royal Masonic Benevolent Institution
- Scarbrough Court, Cramlington, Northumberland, North East England
- 180 kW Peak Heating (UFH System)
- 100 kW Peak DHW (65°C Hot Water).







## **Space Heating**

- 80kW GSHP's for UF space heating
- 2 No. x 40kW NIBE Fighter 1330
- Common Buffer Tank
- Docked with conventional gas boiler
- NIBE "Degree Minute Control".







# **Pre-Heating of DHW**

- 40kW GSHP's for DHW heating
- 1 No. x 40kW NIBE Fighter 1330
- Single Pre-heat Buffer Tank (55°C)
- Pre-heat water pumped to indirect coil within Solar Panel Calorifier
- Calorifier used as pre-heat water
- Boiler increases temperature to 65°C.







## **Ground Energy Resource**

- High heat demand for space heating
- GSHPs active for 5000 Full Load hours/a
- GSHPs providing 400MWh/a to building
- Large cumulative demand from ground
- Required innovation in Ground Energy.

#### Innovation by Overcoming Difficulties

- High annual heat demand from ground
- Restricted site area and depth
- Restricted depth due to mine workings
- No water table due to drained mines...

#### Innovation by Overcoming Difficulties

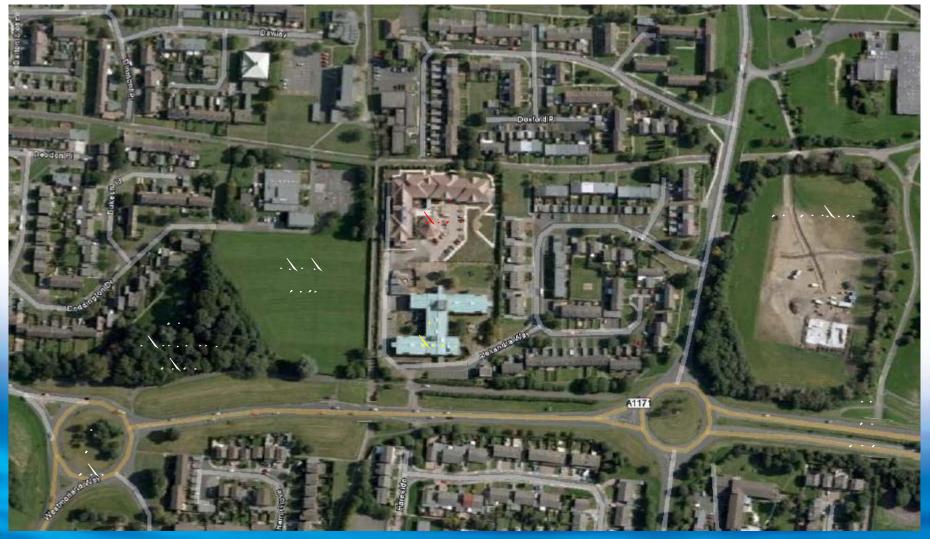
- Accurate mathematical modelling
- Multi-borehole testing to establish two key thermal properties of the ground
- Thermal recharge to the boreholes.



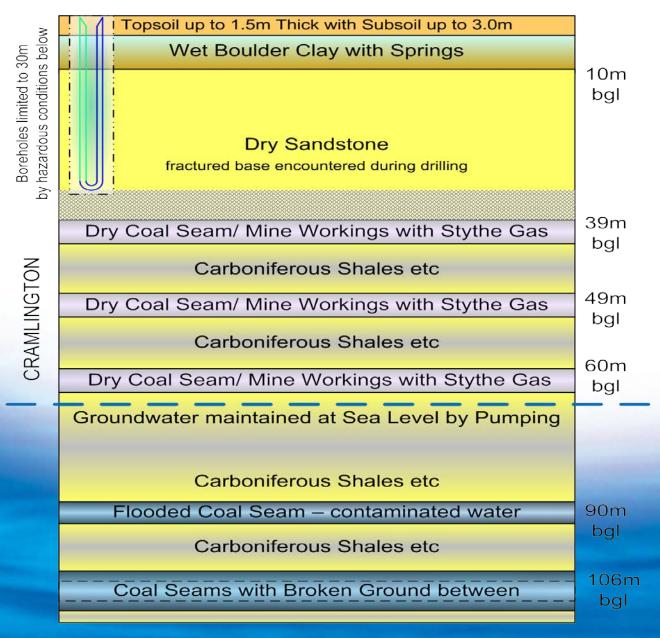
## Mine Workings

- Numerous mineworkings
- Shallowest at 39m below ground level
- Mine fractured ground up to 32m bgl
- Boreholes limited to 30m bgl











#### Mathematical Modelling

- In-house mathematical modelling
- Optimising heat abstraction from ground
- Economic design of borehole field
- Maximise abstraction from each borehole.



## **Collector Field Modelling**

- Requirement to maximise heat abstraction
- Modelling various permutation of collectors.



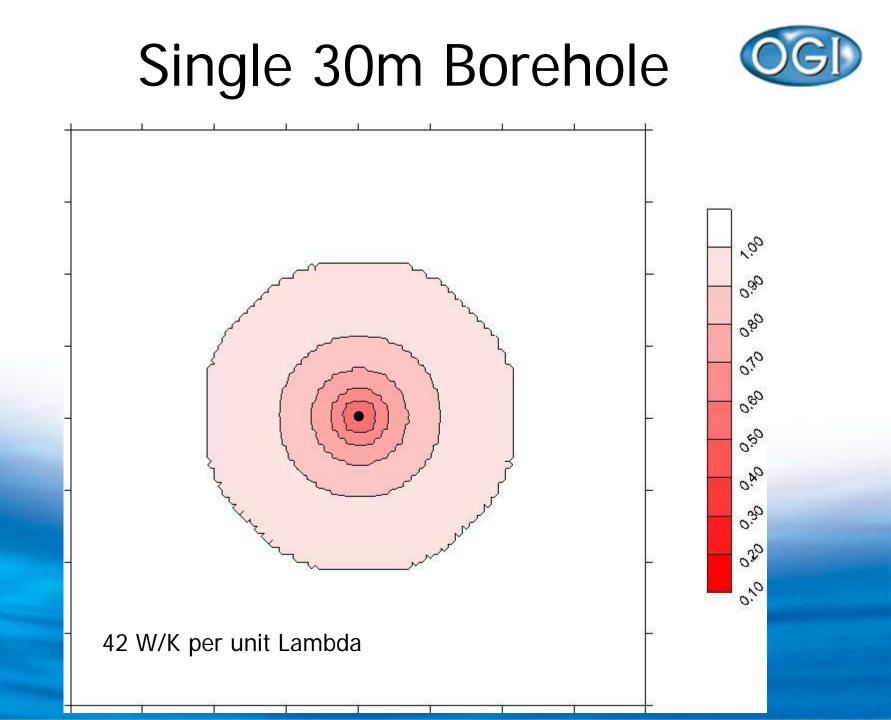
$$\frac{\partial}{\partial x_i} k_{ij} \frac{\partial T}{\partial x_j} = \rho c \frac{\partial T}{\partial t}$$

- *T* = temperature
- $k_{ij}$  = thermal conductivity  $\rho c$  = volumetric heat capacity

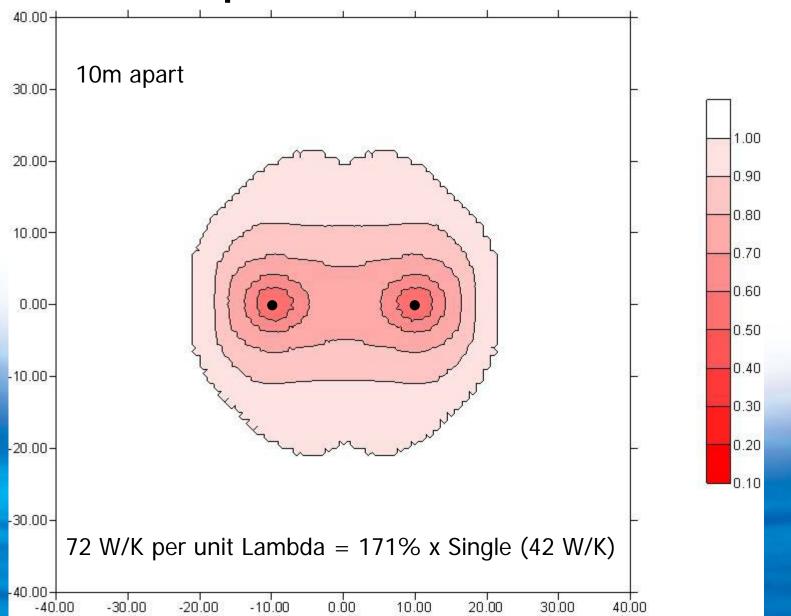
$$x_{i,j}$$
 = space

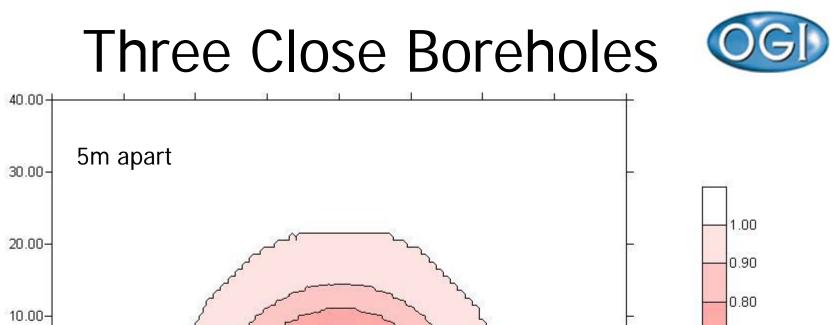
= time t

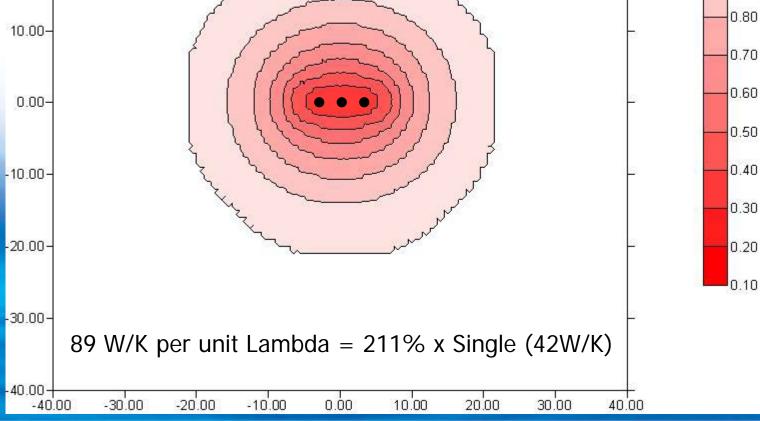




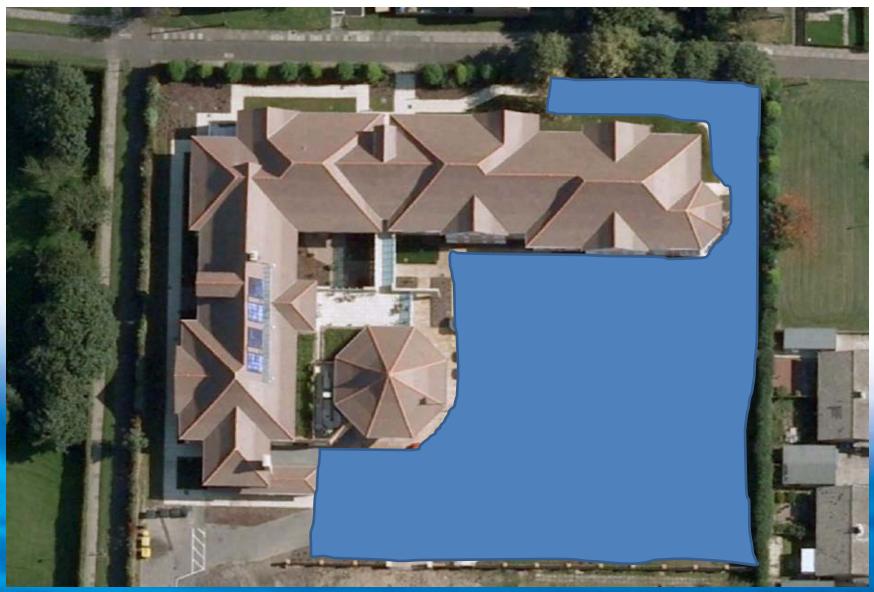
#### Two separated Boreholes

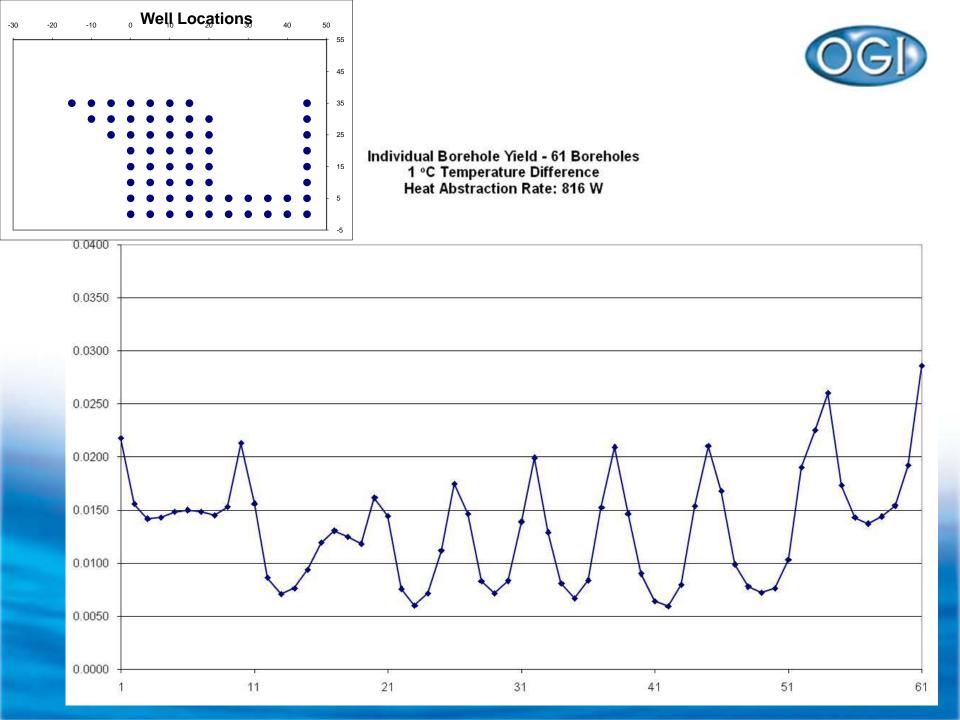


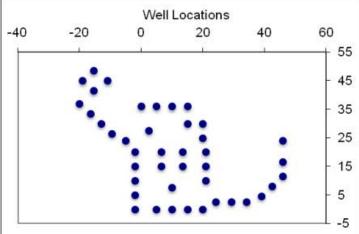






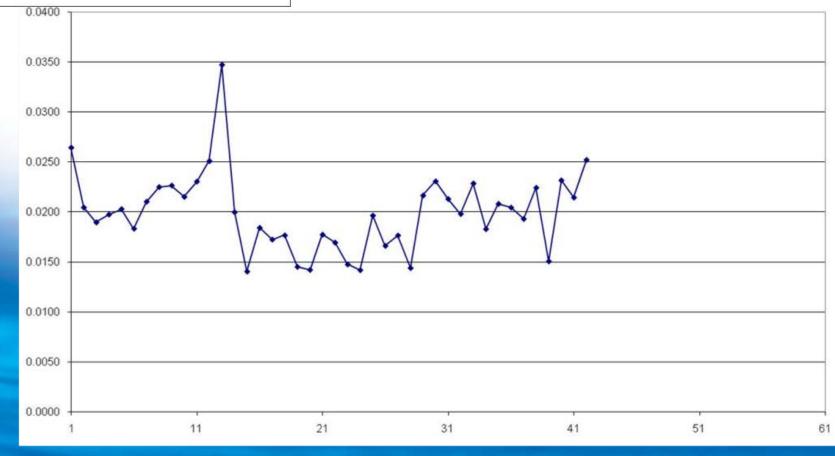






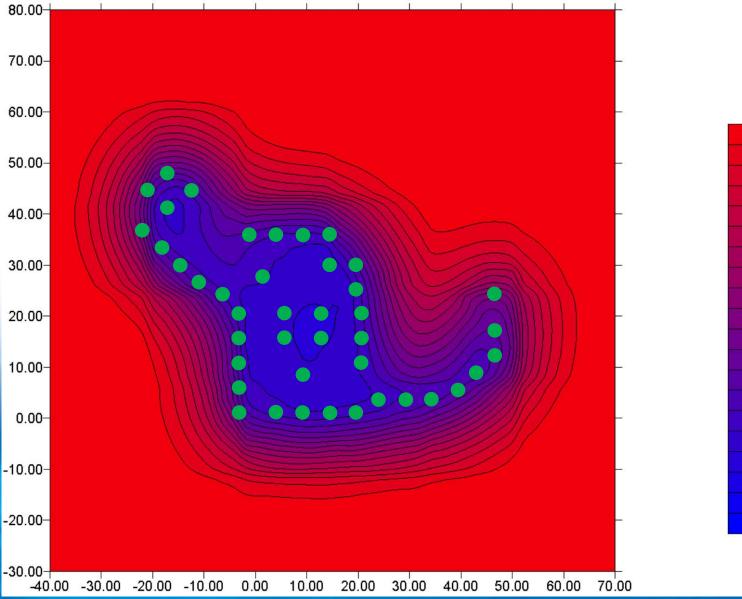


Individual Borehole Yield - 42 Boreholes 1 °C Temperature Difference Total Heat Abstraction Rate: 837 W



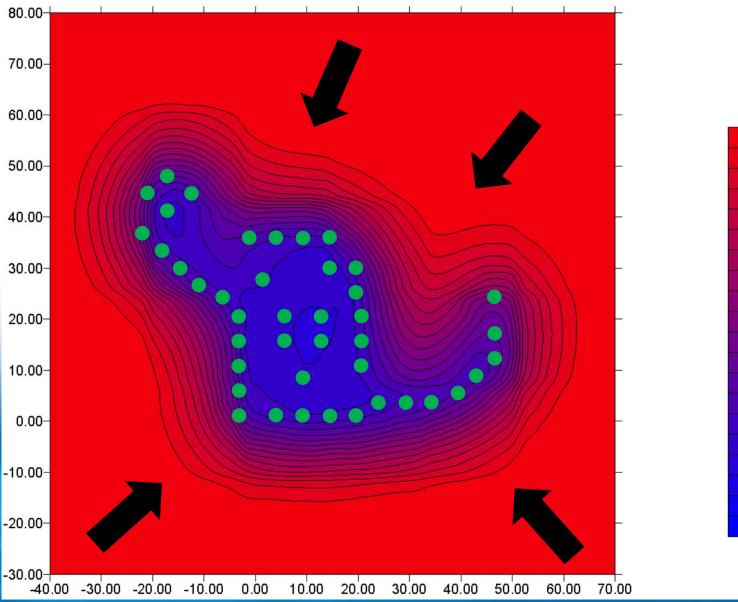


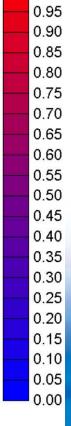
0.95



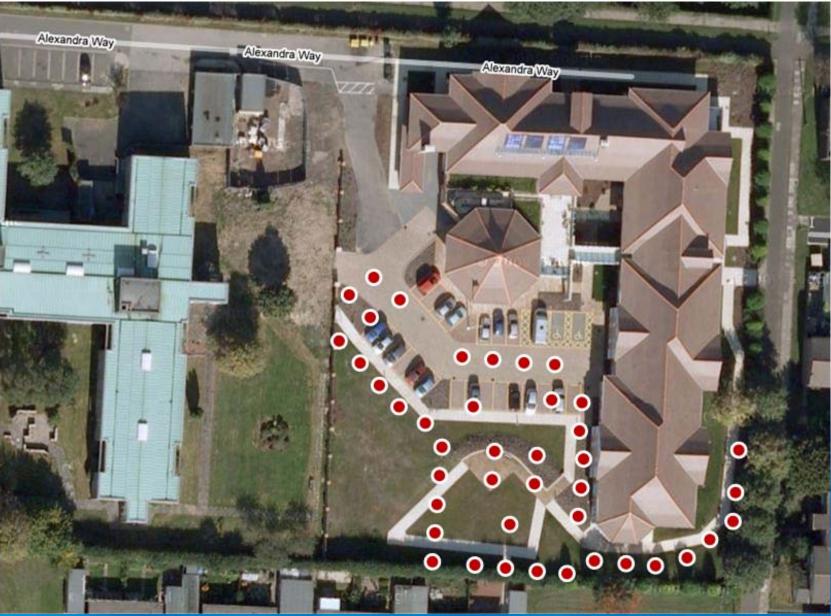
0.90 0.85 0.80 0.75 0.70 0.65 0.60 0.55 0.50 0.45 0.40 0.35 0.30 0.25 0.20 0.15 0.10 0.05 0.00



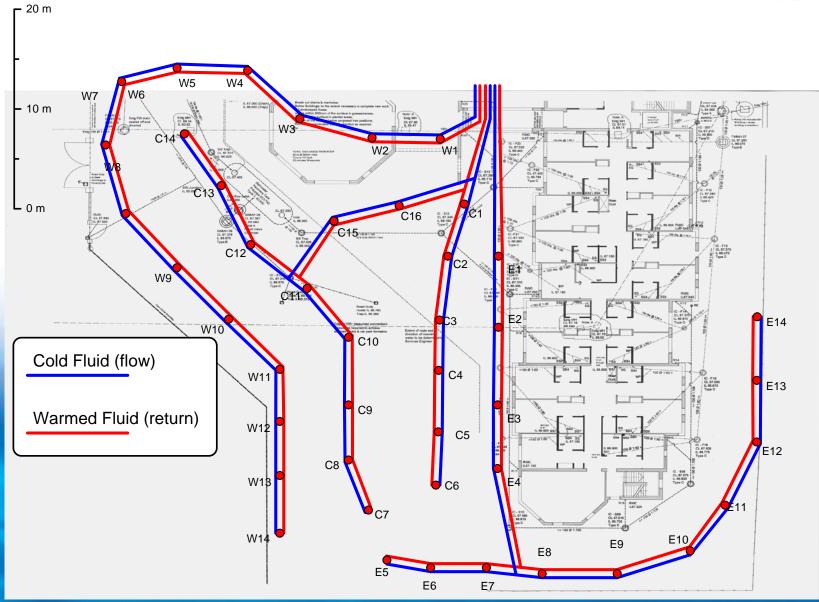










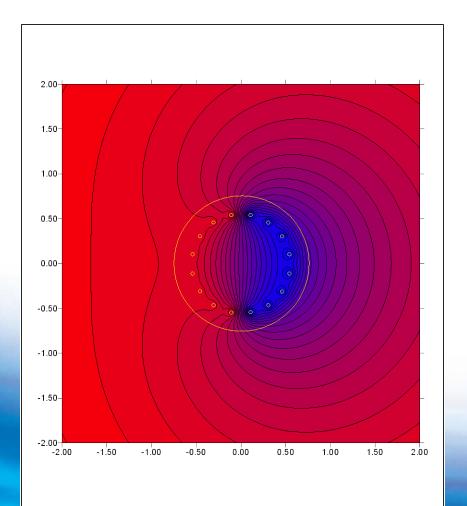


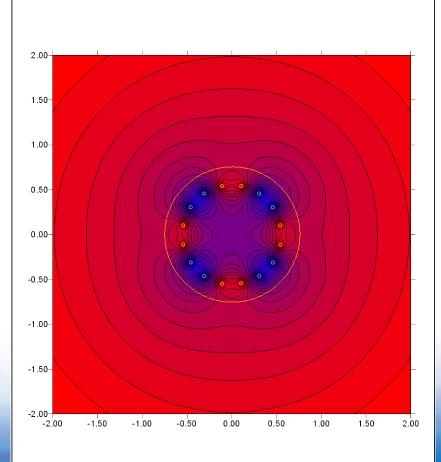


### **Borehole Modelling**

- Requirement to maximise heat abstraction
- Modelling single loop versus double loop.







© OGI Groundwater Specialists Limited 2007		Drawing I	Scale: None pased on: Seftrans 006S
	Project:	Drawing:	
OGD	Energy Piles	Figure 1 (IP/005/002D)	
	Client:	Description:	
Groundwater	Internal Project	Heat Contour	
Specialists	Date:	Drawn by:	Checked by:
·	May 2007	JG	ST

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Groundwater Specialists

Project:	Drawing:	
Energy Piles	Figure 2 (IP/005/003D)	
Client:	Description:	
Internal Project	Heat Contour	
Date:	Drawn by:	Checked by:
May 2007	JG	ST

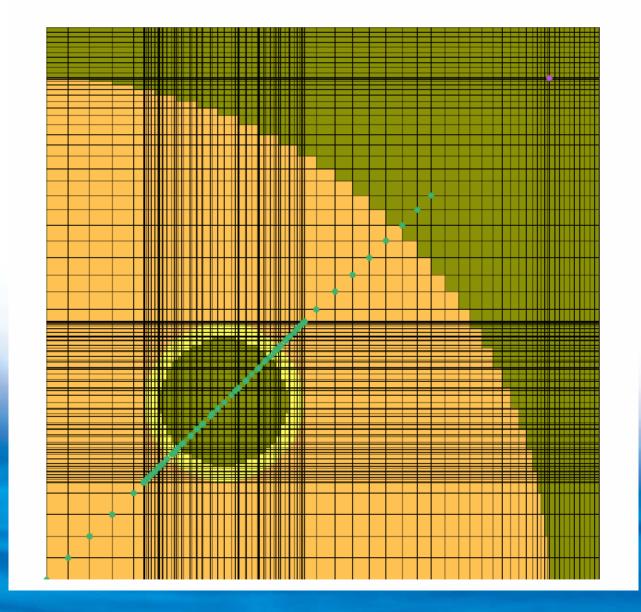
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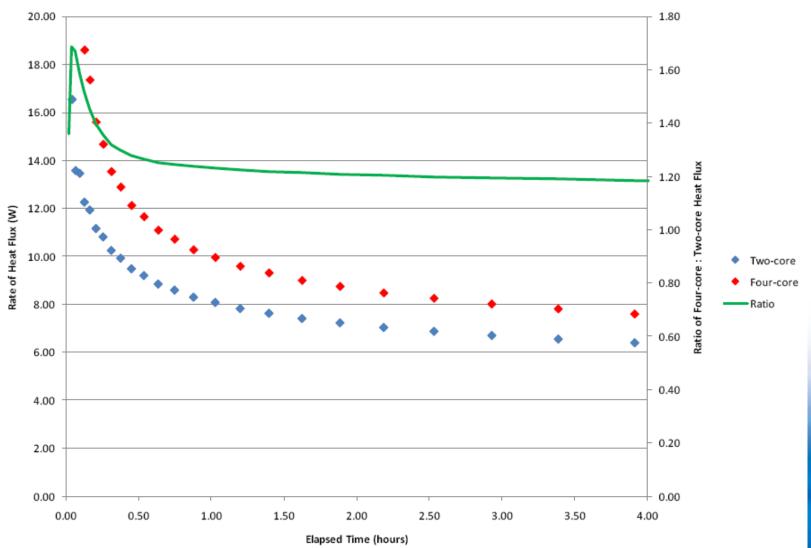


#### Multi-Core Borehole Efficiency





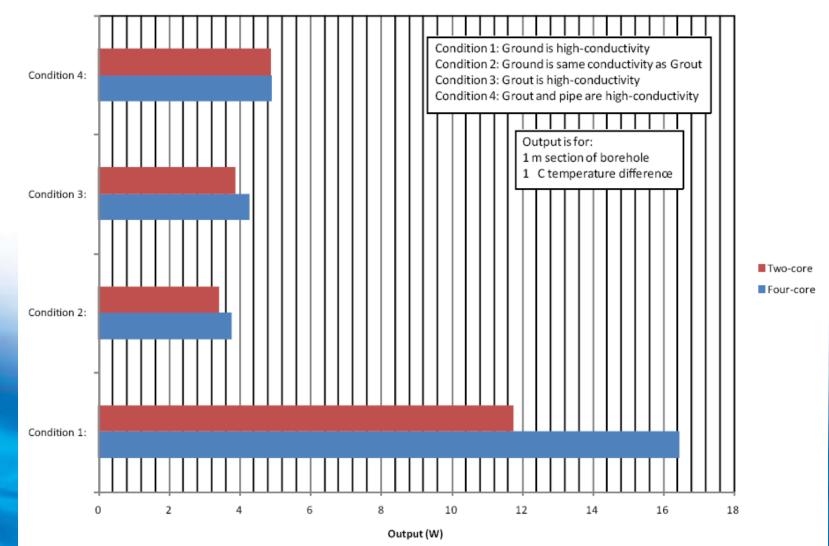




#### **Transient Simulations of Two- and Four-core Boreholes**



#### Heat Output for Four- and Two-core Borehole Arrangements



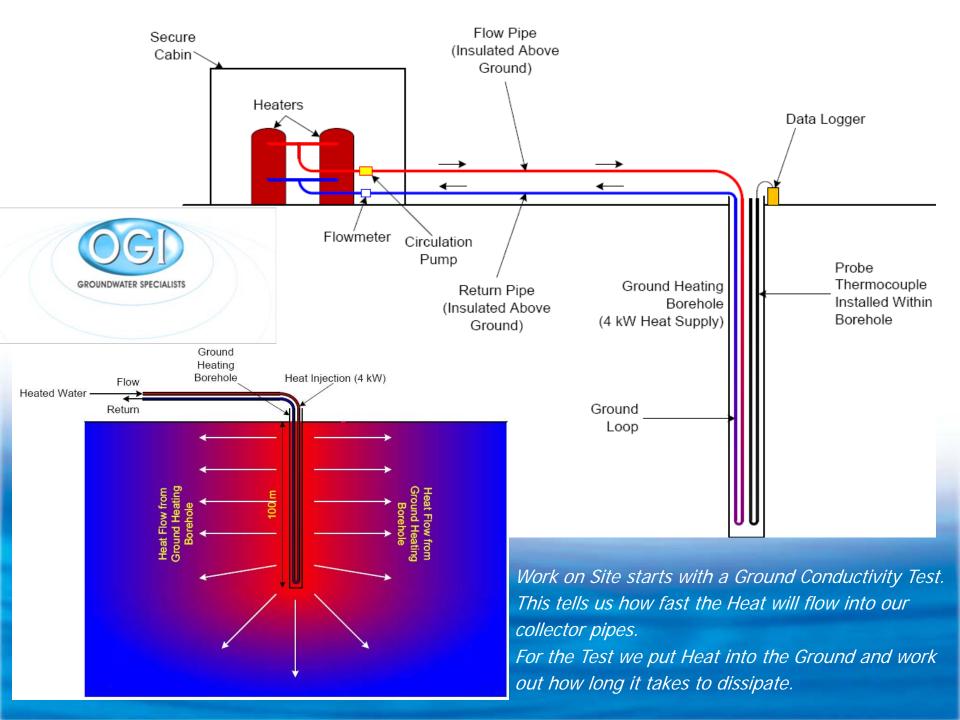


## **Thermal Testing**

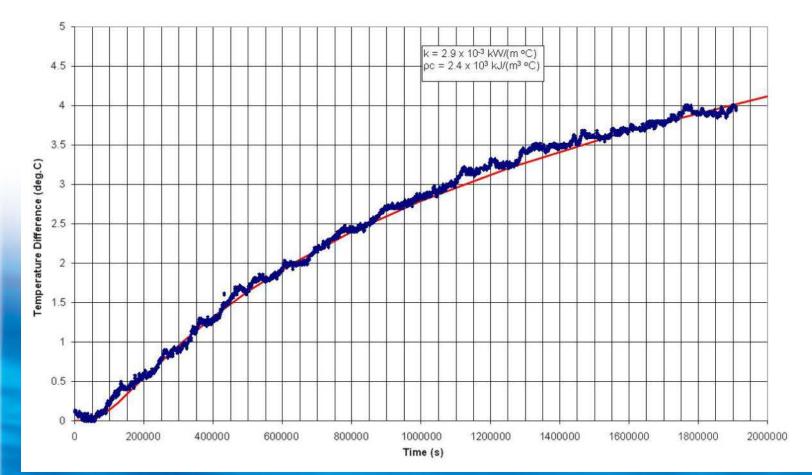
- Heat resource dependent on ground
- Thermal conductivity governs sustainable annual heat resource
- Volumetric heat capacity governs the heat storage of summer recharged heat
- Thermal test using <u>three</u> boreholes
- Observing temperature change at <u>three</u> locations





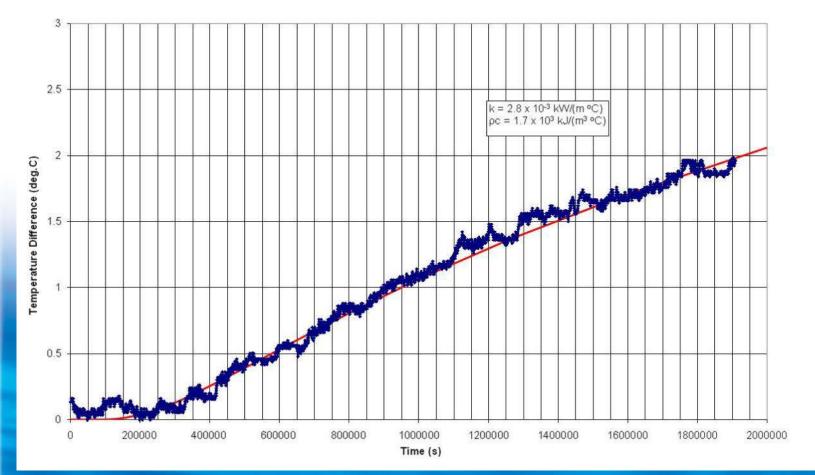






#### Plot of Time Against Temperature for Ground Source Heat Test - MB1





### Plot of Time Against Temperature for Ground Source Heat Test - MB2



### 4.5 <sup>α</sup><u>MB1</u> λ = 2.9 x 10<sup>-3</sup> kW/(m <sup>2</sup>C) ρc = 2.4 x 10<sup>3</sup> kJ/(m<sup>3</sup> <sup>o</sup>C) **Temperature Difference (deg.C)** 3.5 5.2 7 7 7 7 7 7 7 7 7 ALC: $\frac{{}^{D}\mathbf{MB2}}{\lambda = 2.8 \times 10^{-3} \text{ kW/(m °C)}}$ $\rho c = 1.7 \times 10^{3} \text{ kJ/(m ^{3} °C)}$ Part and a 0.5 A .

## Plot of Time Against Temperature for Ground Source Heat Test

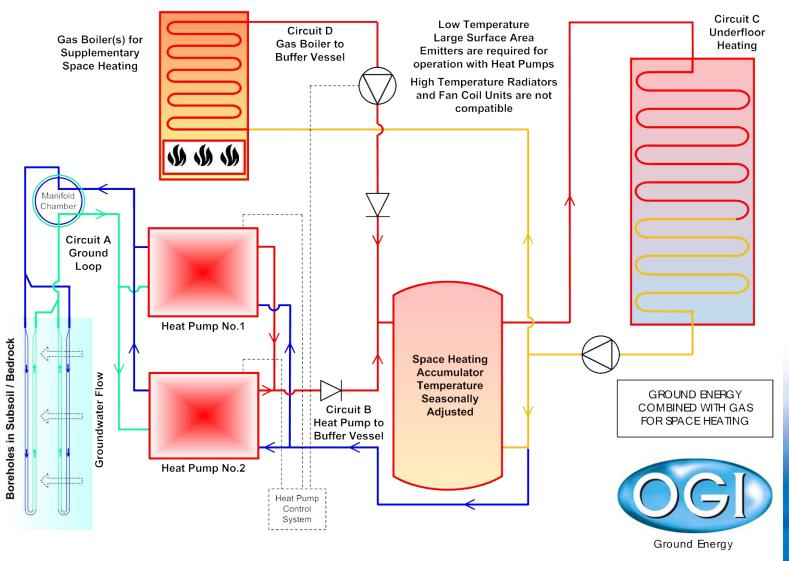
Time (s)



# Integration with Gas Boiler

- GSHP Primary Heat Supply
- Supplemented by 100kW Gas Boiler
- Controlled by NIBE Degree Minute System.



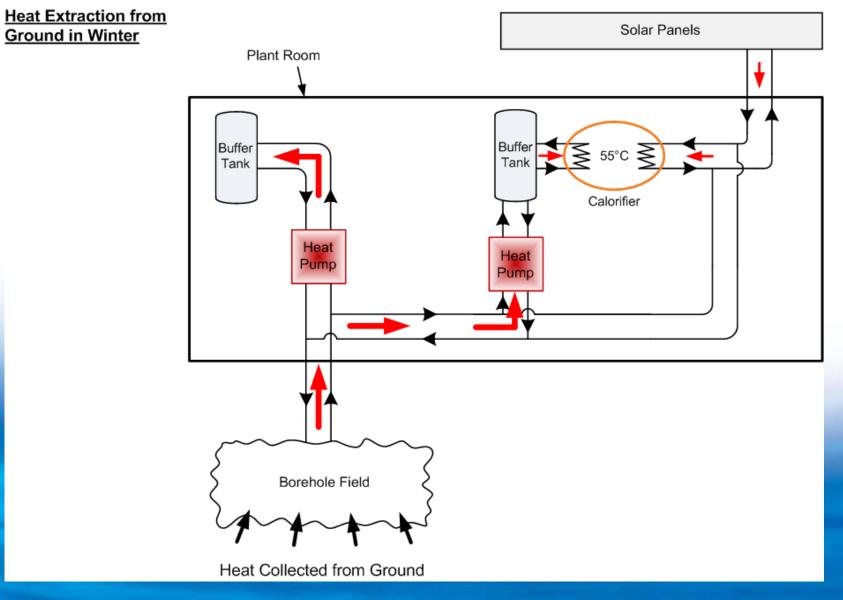




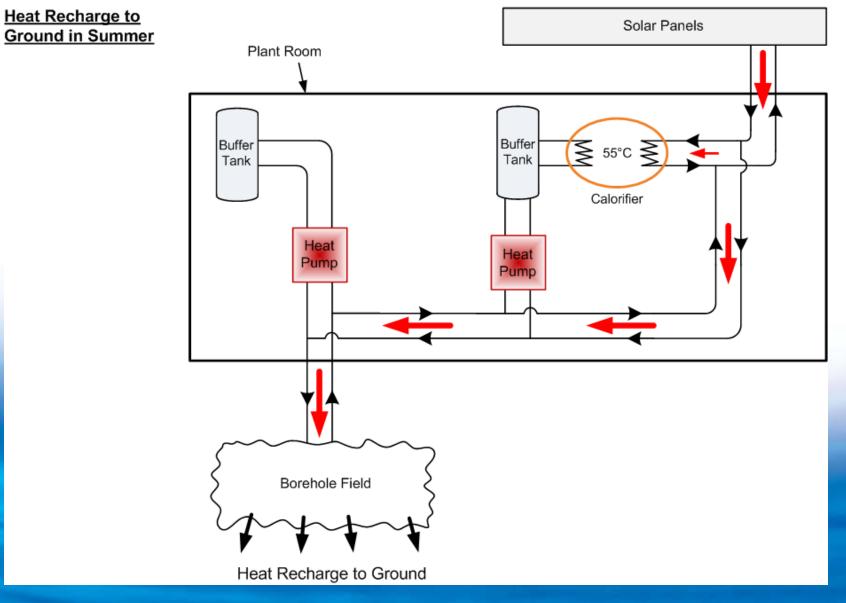
# Integration with Solar Panel

- Solar Panel providing Primary Heat Supply (Riomay Solar Panels)
- GSHP Providing Secondary Heat Supply
- Gas boiler providing Tertiary Heat Supply.

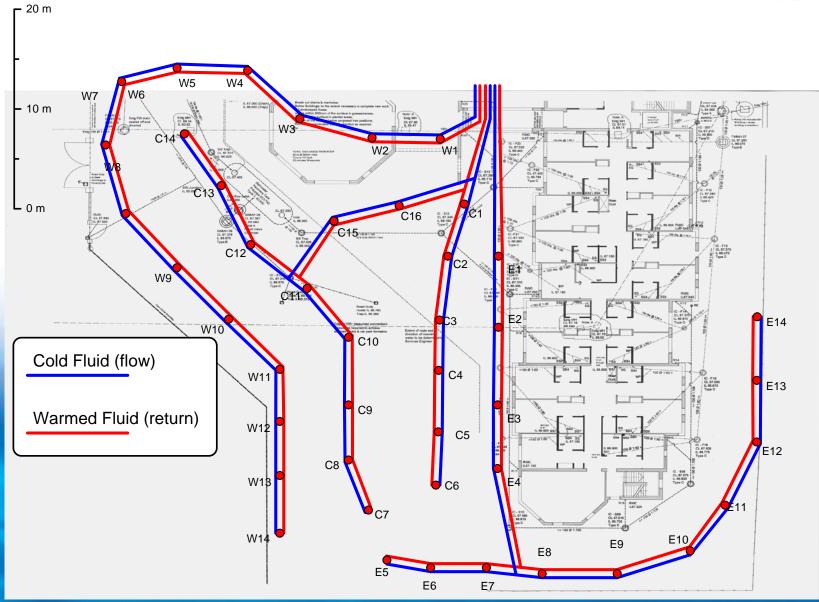














## **Cost Savings**

- GSHP approximately 8 x Gas system to install.
- Running costs approximately 50% of Gas
- Running costs approximately 30% of Oil
- 25 year life on Heat Pumps
- Low maintenance costs
- No deliveries only electrical supply
- Ideally linked to mortgage to reduce CAPEX investment
- Adds asset value to building balance sheet









